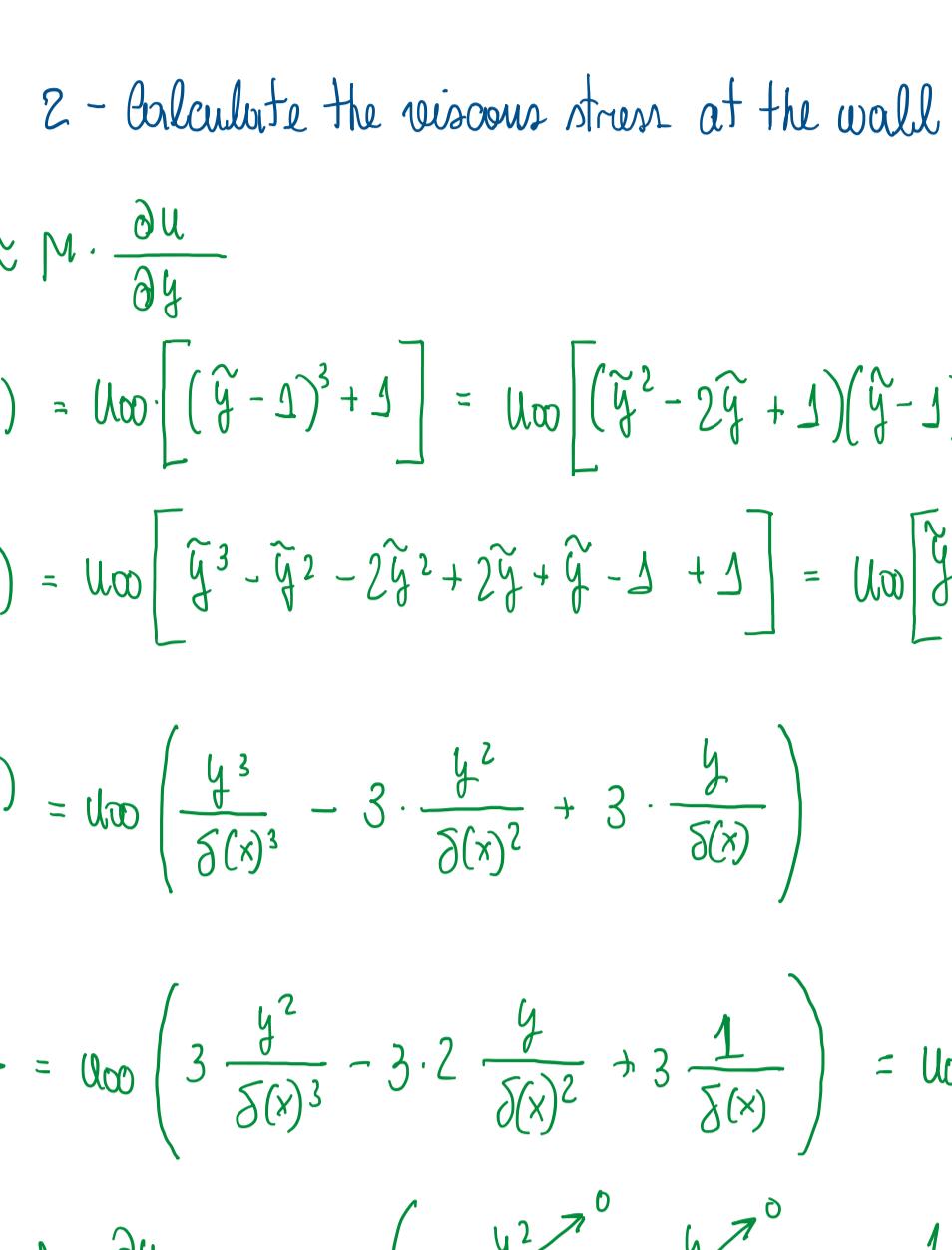


Considering an airfoil which  $U_\infty = 50 \text{ m/s}$  and has an immersed flat plate exhibiting a stall condition. The reference system has  $x$  and  $y$  are parallel and vertical to the flat plate, respectively. In addition, the origin of the reference system coincide with the leading edge of the flat plate. There are two another coordinate frames, 1 and 2. The first establishes that  $x_1$  has a flow attached and with a profile of:

$$u_1(\tilde{y}) = U_\infty \left[ (\tilde{y} - 1)^3 + 1 \right], \quad \tilde{y} = \frac{y}{\delta(x)}$$



where  $\delta(x)$  is the boundary layer thickness. The first part of this exercise has the following question 2. 1 - calculate the Reynolds number:

$$Re = \frac{U_\infty \cdot x}{\nu} = \frac{50 \cdot 2}{15 \cdot 10^{-6}} = 666666.667 = 6,667 \cdot 10^6 \rightarrow \text{TURBULENT}$$

2 - calculate the viscous stress at the wall in  $x_1$ :

$$\tau_{12} \approx \mu \cdot \frac{\partial u}{\partial y}$$

$$u_1(\tilde{y}) = U_\infty \left[ (\tilde{y} - 1)^3 + 1 \right] = U_\infty \left[ (\tilde{y}^2 - 2\tilde{y} + 1)(\tilde{y} - 1) + 1 \right]$$

$$u_1(\tilde{y}) = U_\infty \left[ \tilde{y}^3 - \tilde{y}^2 - 2\tilde{y}^2 + 2\tilde{y} + \tilde{y} - 1 + 1 \right] = U_\infty \left[ \tilde{y}^3 - 3\tilde{y}^2 + 3\tilde{y} \right]$$

$$u_1(\tilde{y}) = U_\infty \left( \frac{\tilde{y}^3}{\delta(x)^3} - 3 \cdot \frac{\tilde{y}^2}{\delta(x)^2} + 3 \cdot \frac{\tilde{y}}{\delta(x)} \right)$$

$$\frac{\partial u}{\partial y} = U_\infty \left( 3 \frac{\tilde{y}^2}{\delta(x)^3} - 3 \cdot 2 \frac{\tilde{y}}{\delta(x)^2} + 3 \frac{1}{\delta(x)} \right) = U_\infty \left( 3 \frac{\tilde{y}^2}{\delta(x)^3} - 6 \frac{\tilde{y}}{\delta(x)^2} + 3 \frac{1}{\delta(x)} \right)$$

$$\tau_w = \mu \cdot \frac{\partial u}{\partial y} = \mu \cdot U_\infty \left( 3 \frac{\tilde{y}^2}{\delta(x)^3} - 6 \frac{\tilde{y}}{\delta(x)^2} + 3 \frac{1}{\delta(x)} \right) = \frac{3 \mu \cdot U_\infty}{\delta(x)}$$

$$\tau_w = 3 \cdot \frac{\mu \cdot U_\infty}{\delta(x)}$$

$$\delta(x) = \frac{0.37 \cdot x}{Re^{1/5}} = 0.37 \cdot \frac{x}{\left( \frac{U_\infty \cdot x^{1/5}}{\nu} \right)^{1/5}} = 0.37 \cdot \frac{x \cdot U_\infty^{1/5} \cdot x^{1/5}}{U_\infty^{1/5} \cdot x^{1/5}} = 0.37 \cdot \frac{x^{1/5} \cdot U_\infty^{1/5}}{U_\infty^{1/5}}$$

$$\tau_w = 3 \cdot \frac{\mu \cdot U_\infty}{0.37 \cdot \frac{U_\infty^{1/5} \cdot x^{1/5}}{U_\infty^{1/5}}} = 3 \cdot \frac{\mu \cdot U_\infty \cdot U_\infty^{1/5}}{0.37 \cdot x^{1/5} \cdot U_\infty^{1/5}} = \frac{3}{0.37} \cdot \frac{\mu \cdot U_\infty^{1/5}}{x^{1/5} \cdot U_\infty^{1/5}}$$

$$\tau_w = \frac{3}{0.37} \cdot \mu \cdot \sqrt{\frac{U_\infty^4}{x^4 \cdot U_\infty^4}} = \frac{3}{0.37} \cdot 18 \cdot 10^{-6} \cdot \sqrt{\frac{50^4}{24 \cdot 15 \cdot 10^6}} = 0.0845 \text{ Pa}$$

The second part of this exercise deals with second reference coordinate  $x_2$ , which the velocity profile is given by:

$$u_2(\tilde{y}) = U_\infty \left[ 2\tilde{y}^6 - 5\tilde{y}^3 + 3\tilde{y}^2 + \tilde{y} \right]$$

3 - Define what motivates separation: The condition for flow separation is indicated by the velocity profile curvature, because for conditions at the wall, the velocity profile has the same sign of the pressure gradient. For the favorable pressure gradient, the curvature is negative throughout the surface and no separation occurs. Hence, to occur separation, the pressure gradient should be positive. When this occurs, the curvature changes sign and separation occurs for  $\frac{\partial u}{\partial y} < 0$ . Hence, the conditions for separation are  $\frac{\partial u}{\partial y} \leq 0$  and  $\frac{\partial p}{\partial x} > 0$ .

$$\frac{\partial^2 u}{\partial y^2} \Big|_w \approx \frac{1}{\mu} \cdot \frac{\partial p}{\partial x} \quad ; \quad U_\infty \left[ 2 \frac{\tilde{y}^4}{\delta(x)^4} - 5 \cdot \frac{\tilde{y}^3}{\delta(x)^3} + 3 \cdot \frac{\tilde{y}^2}{\delta(x)^2} + \frac{\tilde{y}}{\delta(x)} \right]$$

$$\frac{\partial u}{\partial y} = U_\infty \left[ 2 \cdot 4 \frac{\tilde{y}^3}{\delta(x)^4} - 5 \cdot 3 \frac{\tilde{y}^2}{\delta(x)^3} + 3 \cdot 2 \frac{\tilde{y}}{\delta(x)^2} + \frac{1}{\delta(x)} \right]$$

$$\frac{\partial u}{\partial y} = U_\infty \left[ 8 \cdot \frac{\tilde{y}^3}{\delta(x)^4} - 15 \cdot \frac{\tilde{y}^2}{\delta(x)^3} + 6 \cdot \frac{\tilde{y}}{\delta(x)^2} + \frac{1}{\delta(x)} \right] = \frac{U_\infty}{\delta(x)}$$

$$\frac{\partial^2 u}{\partial y^2} = U_\infty \left[ 8 \cdot 3 \frac{\tilde{y}^2}{\delta(x)^4} - 15 \cdot 2 \frac{\tilde{y}}{\delta(x)^3} + 6 \frac{1}{\delta(x)^2} \right]$$

$$\frac{\partial^2 u}{\partial y^2} = U_\infty \left[ 24 \frac{\tilde{y}^2}{\delta(x)^4} - 30 \frac{\tilde{y}}{\delta(x)^3} + 6 \frac{1}{\delta(x)^2} \right] = 6 \cdot \frac{U_\infty}{\delta(x)^2}$$

$$\frac{\partial^2 u}{\partial y^2} = - \frac{1}{\mu} \cdot \frac{\partial p}{\partial x} = 6 \cdot \frac{U_\infty}{\delta(x)^2}$$

$$\delta(x) = 0.37 \cdot \frac{x}{Re^{1/5}} = 0.37 \cdot \frac{2}{666666.667^{1/5}} = 0.0319 \text{ m}$$

$$\frac{\partial p}{\partial x} = 6 \cdot \frac{18 \cdot 10^{-6} \cdot 50}{0.0319^2} = 5.307 \rightarrow \text{Positive}$$

Hence, the curvature is positive, which confirms that no occurs separation, but  $\frac{\partial p}{\partial x}$  is also positive. This means that the pressure gradient adverse, but without separation.

There are instabilities on the flow, but not enough to trigger separation. The next part of the exercise proposes a third coordinate, which is  $x_3$  that has the following velocity profile:

$$u_3(\tilde{y}) = U_\infty \left[ 4\tilde{y}^4 - 11\tilde{y}^3 + 9\tilde{y}^2 - \tilde{y} \right]$$

5 - calculate the viscous stress at the wall

$$u_3\left(\frac{y}{\delta(x)}\right) = U_\infty \left[ 4 \cdot \frac{\tilde{y}^4}{\delta(x)^4} - 11 \cdot \frac{\tilde{y}^3}{\delta(x)^3} + 9 \cdot \frac{\tilde{y}^2}{\delta(x)^2} - \frac{\tilde{y}}{\delta(x)} \right]$$

$$\frac{\partial u}{\partial y} = U_\infty \left[ 4 \cdot 4 \cdot \frac{\tilde{y}^3}{\delta(x)^4} - 11 \cdot 3 \cdot \frac{\tilde{y}^2}{\delta(x)^3} + 9 \cdot 2 \cdot \frac{\tilde{y}}{\delta(x)^2} - \frac{1}{\delta(x)} \right]$$

$$\frac{\partial u}{\partial y} = U_\infty \left[ 16 \cdot \frac{\tilde{y}^3}{\delta(x)^4} - 33 \cdot \frac{\tilde{y}^2}{\delta(x)^3} + 18 \cdot \frac{\tilde{y}}{\delta(x)^2} - \frac{1}{\delta(x)} \right]$$

$$\tau_w = U_\infty \cdot \frac{\partial u}{\partial y} = U_\infty \cdot \left[ 16 \cdot \frac{\tilde{y}^3}{\delta(x)^4} - 33 \cdot \frac{\tilde{y}^2}{\delta(x)^3} + 18 \cdot \frac{\tilde{y}}{\delta(x)^2} - \frac{1}{\delta(x)} \right]$$

$$\tau_w = - \frac{U_\infty}{\delta(x)} = - \frac{U_\infty}{0.0319} = - 1.8 \cdot 10^{-6} \cdot 50 \cdot \frac{1}{0.0319}$$

$$\tau_w = - 9 \cdot 10^{-4} \frac{1}{0.0319} = 0.028 \text{ Pa}$$

If the average  $C_p$  at front and at the back are given by  $\bar{C}_p = -0.5$  and  $\bar{C}_p = -1.5$ , respectively using:

$$\bar{C}_p = - \frac{1}{S} \int_C \bar{C}_p \cdot \bar{n} \cdot dS$$

and the flat plate assume an inclination  $\alpha_s = 140^\circ$ .

$$\bar{C}_p = - \frac{1}{S} \int_C \bar{C}_p \cdot \bar{n} \cdot dS = - \frac{1}{S} \int_C \bar{C}_p \cdot \bar{n} \cdot \cos \alpha_s \cdot dL_2$$

$$\bar{C}_p = - \frac{1}{S} \int_C \bar{C}_p \cdot \bar{n} \cdot dS = - \frac{1}{S} \int_C \bar{C}_p \cdot \bar{n} \cdot \cos \alpha_s \cdot dL_2 = \bar{C}_{p,f} \cdot \bar{C}_{p,b} \cdot \cos \alpha_s$$

$$\bar{C}_p = \bar{C}_{p,f} + \bar{C}_{p,b} = \bar{C}_{p,f} \cdot \bar{C}_{p,b} \cdot \cos \alpha_s - \bar{C}_{p,f} \cdot \bar{C}_{p,b} \cdot \cos \alpha_s$$

$$\bar{C}_p = 1500 \cdot (-0.5) \cdot \cos 140^\circ \cdot 2 - 1500 \cdot (-1.5) \cdot \cos 140^\circ \cdot 2$$

$$\bar{C}_p = - 1455.444 + 4366.321$$

$$\bar{C}_p = 2910.887 \text{ N/m}$$

$$\bar{C}_p = 725.766 \text{ N/m}$$

Finally, it is possible to visualize that as soon the flat plate exhibits an angle, it is created a drag and lift components due to the pressure distribution over the surface. Hence it is just to derive the pressure coefficient about the area by the following equations:

$$F = - \frac{1}{S} \int_C \bar{C}_p \cdot \bar{n} \cdot dS$$

$$F = - \frac{1}{S} \int_C \bar{C}_p \cdot \bar{n} \cdot dS = - \frac{1}{S} \int_C \bar{C}_p \cdot \bar{n} \cdot \int_{L_1}^{L_2} dL_2 \rightarrow \text{per unit length}$$

$$F = - \frac{1}{S} \int_C \bar{C}_p \cdot \bar{n} \cdot dS = - \frac{1}{S} \int_C \bar{C}_p \cdot \bar{n} \cdot \int_{L_1}^{L_2} dL_2 \rightarrow \text{per unit length}$$

However, there are two forces on the flat plate, both are opposed to the pressure distribution, thus:

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \sin \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \sin \alpha_s \cdot dL_2$$

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2$$

the  $\sin \alpha_s$  and  $\cos \alpha_s$  change due to the orientation of vector  $\bar{n}$  respective to the vector  $i$  and  $j$ , if pointing to the positive direction, the product  $\bar{n} \cdot i$  and  $\bar{n} \cdot j$  will be positive. Conversely, when  $\bar{n}$  is at opposite direction, the products  $\bar{n} \cdot i$  and  $\bar{n} \cdot j$  are negative.

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2$$

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2$$

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2$$

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2$$

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2$$

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2$$

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2$$

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2$$

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2$$

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2$$

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2$$

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2$$

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2$$

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2 - \bar{C}_{p,b} \cdot \bar{C}_p \cdot \cos \alpha_s \cdot dL_2$$

$$\bar{C}_p = \bar{C}_{p,f} \cdot \bar{C}_p \$$